

WRAPPING APPARATUS

1 The present invention relates to a wrapping apparatus as defined in the preamble of claim 1.

In prior art, a wrapping apparatus for winding a wrapping film about an article to be packaged is known. The wrapping apparatus comprises a film dispenser arranged to revolve along a circular endless track at a constant velocity about the article to be wrapped. The film dispenser comprises a frame, supporting elements for supporting a film roll on the frame, and a pre-stretching device. The pre-stretching device comprises a first pre-stretching roller rotatably mounted on the frame with bearings at both ends to receive the film from the film roll, and a second pre-stretching roller rotatably mounted on the frame with bearings at both ends and disposed in a position parallel to and at a distance from the first pre-stretching roller. The pre-stretching rollers are coupled together via a regular transmission so that their circumferential velocities differ from each other, the pre-stretching of the film thus occurring within the film portion between the pre-stretching rollers as a result of their different circumferential velocities. The film dispenser further comprises a pendulum roller disposed after the second pre-stretching roller in the direction of film movement to receive the pre-stretched film from the second pre-stretching roller. The pendulum roller is spring-loaded with a spring acting against the drawing direction of the film web. The film dispenser further comprises a deflecting roller mounted by both ends with bearings on the frame, in a position parallel to the pre-stretching rollers and the pendulum roller, the film web coming from the pendulum roller being passed over the deflecting roller to the article to be wrapped.

Typically, the product to be wrapped is a pallet and its cross-section is of a rectangular form.

Therefore, the feed rate (i.e. the drawing speed of the film in relation to the film dispenser) at which the film is passed from the film dispenser onto the article varies as the film dispenser is revolving at a constant speed about the article. As the film is stretched between the rollers, the film portion after the pre-stretching rollers is strained to a tension that is proportional to the speed difference between the rollers, to the drawing velocity, to the thickness and width of the film and to the internal elongation properties of the film. As a result of the quadrangular form of the object being wrapped, e.g. a pallet, the drawing velocity of the film varies continuously, producing variation in film tension.

15 *SM 27* To eliminate the variation of tension, prior-art apparatus use a sensor connected to the pendulum roller to detect the degree of film tension. The film tension detected by the sensor is used as a basis to control the speed of rotation of either the motor driving the film roll, as in specification EP 0 936 141 A1, the motor driving the posterior pre-stretching roller as seen in the direction of film motion, as in specification US 5,123,230 A, or the drive motor of some other roller used to draw the film from the film roll, as in specification WO 93/24373. These arrangements are designed to eliminate the variation in film tension resulting from the angular shape of the article to be wrapped and to achieve a constant film tension.

30 The problem with prior-art apparatus is that they are complex and expensive as they need tension detectors and drive motors controlled on the basis of these to drive the rollers and/or the film roll.

Ins B3 *B3 7* The object of the invention is to eliminate the above-mentioned problems.

A specific object of the invention is to disclose an apparatus having a construction that is sim-

pler, cheaper, more reliable and less susceptible to failure than prior-art devices.

A further object of the invention is to disclose an apparatus in which the drawing velocity and
 5 tension of the film can be held substantially constant regardless of the varying draw and drawing velocity of the film caused by the non-circular shape of the article to be wrapped and in which this is achieved using a completely mechanical construction.

~~The wrapping apparatus of the invention is characterized by what is presented in claim 1.~~

According to the invention, the supporting elements are mounted on the frame with bearings permitting free rotation so that the film roll supported
 15 by them is freely rotatable. The pre-stretching rollers are mutually coupled and mounted with bearings on the frame so as to be freely rotatable. The pendulum roller and the spring force of the spring are so adapted that the pendulum roller forms between a second pre-stretching roller and a deflecting roller a
 20 bend acting as a film supply which contains a varying amount of film, depending on the prevailing draw of the film, to keep the drawing velocity and tension of the film substantially constant at the pre-stretching
 25 rollers regardless of the variation in the draw and velocity of the film in relation to the film dispenser that is caused by the shape of the article to be wrapped. The spring is so designed that it tends to keep a predetermined maximum amount of film in the
 30 film supply.

When the drawing velocity of the film increases, the film supply delivers film so that the drawing velocity at the pre-stretching rollers remains substantially constant, and when the drawing velocity
 35 of the film decreases, the film supply is replenished. The invention makes it possible to maintain a constant film tension throughout the wrapping process. The film

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drive means, such as a motor, for rotating the film roll.

Further, mounted on the frame 3 is a pre-stretching device 6, 7 comprising a first pre-stretching roller 6, which is rotatably mounted on the frame 3 with bearings at both ends and receives the film from the film roll 5. The pre-stretching device further comprises a second pre-stretching roller 7, which is rotatably mounted on the frame 3 with bearings at both ends, in a position parallel to and at a small distance from the first pre-stretching roller 6. The pre-stretching rollers 6, 7 are supported on the frame by bearings permitting free rotation, i.e. without any drive means, yet so that the pre-stretching rollers are mutually engaged via a regular gear transmission 8. Fig. 5 illustrates a gear transmission in which a first gear 21 is attached to the first pre-stretching roller 6 and a second gear 22 to the second pre-stretching roller 7 and the gears are in direct driving mesh with each other. Thus, the two pre-stretching rollers have different circumferential velocities. To achieve pre-stretching of the film, the transmission ratio of the gear transmission is so selected that the circumferential speed of the second pre-stretching roller 7, which is the posterior one as seen in the direction of film movement, is higher (e.g. about 10% higher) than the circumferential speed of the first pre-stretching roller 6, the pre-stretching of the film thus occurring within the film portion between the pre-stretching rollers as a result of the different circumferential speeds of the pre-stretching rollers.

The film dispenser 1 further comprises a pendulum roller 9, which is disposed after the second pre-stretching roller 7 as seen in the direction of film movement to receive the pre-stretched film from the second pre-stretching roller 7. The pendulum

roller 9 is loaded by a spring 10 against the drawing direction of the film F. The pendulum roller 9 comprises a diverting element 12, which is parallel to the pre-stretching rollers 6, 7 and the deflecting roller 11, the film F being passed over the diverting element. The pendulum arms 13 are connected to either end of the diverting element transversely to the longitudinal direction of the diverting element. The turn arbor 14 is fastened to the pendulum arms 13 and mounted with bearings on the frame 3. Attached to the turn arbor 14 is a lever 15. The lever 15 is provided with a fastening element 16 for holding the spring 10. The spring is a helical spring whose one end is connected to the lever 15 while its other end is connected to the frame 3.

The film dispenser further comprises a deflecting roller 11, which is mounted on the frame 3 with bearings at each end, in a position parallel to the pre-stretching rollers 6, 7 and the pendulum roller 9, the film F coming from the pendulum roller 9 being passed over the deflecting roller to the object to be wrapped.

The film F is so threaded over the rollers that the first pre-stretching roller 6, the pendulum roller 9 and the deflecting roller 11 are in contact with the first side 19 of the film F while the second pre-stretching roller 7 is in contact with the second side 20 of the film.

The pendulum roller 9 and the spring force of the spring 10 are so adapted that the pendulum roller 9 creates a bend between the second pre-stretching roller 7 and the deflecting roller 11, said bend acting as a film supply that contains a varying amount of film depending on the prevailing draw of the film F, to keep the drawing velocity and tension of the film substantially constant regardless of the variation in the draw and drawing velocity of the film that is

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caused by the shape of the article to be wrapped. As a result of the action of the pendulum roller 9, the drawing velocity over the rollers becomes constant, the variation of tension being thus eliminated.

5 The spring 10 is so designed that it tends to keep the film supply full. When the rate of film consumption increases, the drawing velocity at the rollers is not substantially increased because the film supply first delivers the film it contains. When the
10 rate of consumption falls, the film supply is replenished and the velocity and therefore the film tension at the rollers remain constant. The film supply only starts delivering film when the film feed rate exceeds a certain value, whereupon the film supply begins de-
15 livering film and the tension assumes a predetermined level. When the film feed rate exceeds a certain level, the film supply will deliver film because the tension becomes greater than the spring force.

As can be seen from Fig. 4, the apparatus
20 comprises limit elements 17, 18 for limiting the deflection angle of the pendulum roller 9 to a predetermined magnitude; in this embodiment, the maximum deflection angle of the pendulum arm 13 between the extreme positions is 60°. The limit elements 17, 18 com-
25 prise a first limit element 17 determining a first extreme position I of the pendulum roller 9, in which the film supply formed by it contains a maximum amount of film, and a second limit element 18 determining a second extreme position II of the pendulum roller 9,
30 in which the film supply formed by it contains a minimum amount of film.

Fig. 6 presents a preferred geometric layout of the rollers 7, 9 and 11. In designing the layout, it is important that the film supply should contain a
35 sufficient amount of film to eliminate the variation in film tension caused by the drawing velocity of the film, and that the film tension remains substantially

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the same regardless of the amount of film in the film supply.

The maximum deflection angle of the pendulum arm 13 between the extreme positions I and II is 60° .

5 The distance between the swing axis 14 of the pendulum roller 9 and the center axis of the deflecting roller 11 is designated as x . In that case, the distance between the center axis of the second pre-stretching roller 7 and the diverting element 12 of the pendulum
10 roller 9 is equal to $3.04 \cdot x$. The distance between the center axis of the diverting element 12 of the pendulum roller 9 and the swing axis 14 of the pendulum roller 9 equals $1.31 \cdot x$. The distance between the center axis of the deflecting roller 11 and the center
15 axis of the second pre-stretching roller 7 equals $1.73 \cdot x$. The distance between the swing axis 14 of the pendulum roller 9 and the center axis of the second pre-stretching roller 7 equals $2.62 \cdot x$.

The angle formed between the first extreme
20 position I of the pendulum roller and the center axis of the second pre-stretching roller 7 with the swing axis 14 of the pendulum roller 9 at the angle vertex equals 95.4° . The angle formed between the first extreme position I of the pendulum roller and the center
25 axis of the deflecting roller 11 with the swing axis 14 of the pendulum roller 9 at the angle vertex equals 116.6° .

In a certain design, the distance x between the swing axis 14 of the pendulum roller 9 and the
30 center axis of the deflecting roller 11 equals 105.4 mm. In this case, the distance between the center axis of the second pre-stretching roller 7 and the center axis of the diverting element 12 of the pendulum roller 9 equals 320.3 mm. The distance between the
35 center axis of the diverting element 12 of the pendulum roller 9 and the swing axis 14 of the pendulum roller 9 equals 137.6 mm. The distance between the

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center axis of the deflecting roller 11 and the center axis of the second pre-stretching roller 7 equals 182.3 mm. The distance between the swing axis 14 of the pendulum roller 9 and the center axis of the second pre-stretching roller 7 equals 276.5 mm. The length R of the lever 15 (measured from the swing axis 14 of the pendulum roller 9 to the fastening point of the spring 10) equals 60 mm. The spring constant of the spring 10 is 1.4 N/mm.

Fig. 7 represents the circular track 2 of the film dispenser and a rectangular package P to be wrapped placed inside it, the dimensions of the package corresponding to a standard pallet measuring 800 mm x 1200 mm. The track circle 2 has been divided into sections at 10-degree intervals and the length of the film span has been calculated for each section. Based on the length of the film span, it is possible to calculate the instantaneous film tension for each point in Fig. 7, the film properties, film dispenser speed and the film track design according to Fig. 6 in the film dispenser being known.

Fig. 8 presents a curve representing the film tension in units N at different points along the track, calculated on the basis of the above-mentioned starting values. Fraction line FJ represents film tension when no pendulum roller is used. Fraction line FJv represents film tension when a pendulum roller 9 is used. Fig. 8 shows clearly that the variation in film tension is considerably smaller when a pendulum roller is used, as compared with a situation where no pendulum roller is used.

The invention is not restricted to the examples of its embodiments described above; instead, many variations are possible within the scope of the inventive idea defined in the claims.